

**COVENANT UNIVERSITY  
NIGERIA**

*TUTORIAL KIT  
OMEGA SEMESTER*

**PROGRAMME: BUILDING  
TECHNOLOGY**

**COURSE: BLD 224**

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## BLD 224: STRUCTURAL THEORY AND STRENGTH OF MATERIAL II

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1. A steel bar of rectangular cross-section, 3 cm by 2 cm, carries an axial load of 30 kN. Estimate the average tensile stress over a normal cross-section of the bar.
2. A cylindrical block is 30 cm long and has a circular cross-section 10 cm in Diameter. It carries a total compressive load of 70 KN, and under this load it Contracts by 0.02 cm. Estimate the average compressive stress over a normal Cross-section and the compressive strain.
3. Explain the following terms: Linear of proportionality, Elastic limit and Breaking point.
4. A steel bolt, 2.50 cm in diameter, carries a tensile load of 40 kN. Estimate the average tensile stress at the section  $a$  and at the screwed section  $b$ , where the diameter at the root of the thread is 2.10 cm.
5. A tensile test is carried out on a bar of mild steel of diameter 2 cm. The bar Yields under a load of 80 kN. It reaches a maximum load of 150 KN, and breaks finally at a load of 70 kN.  
Estimate:
  - (i) The ultimate tensile stress;
  - (ii) The tensile stress at the yield point;
  - (iii) The average stress at the breaking point, if the diameter of the fractured neck is 1 cm.
- 6 Differentiate between brittle and ductile material and give examples
- 7 The piston of a hydraulic ram is 40 cm diameter, and the piston rod 6 cm diameter. The water pressure is  $1 \text{ MN/m}^2$ . Estimate the stress in the piston rod and the elongation of a length of 1 m of the rod when the piston is under Pressure from the piston-rod side. Take Young's modulus as  $E = 200 \text{ GN/m}^2$ .
- 8 A bar of cross-section 2.25 cm by 2.25 cm is subjected to an axial pull of 20 kN. Calculate the normal stress and shearing stress on a plane the normal to which makes an angle of  $60^\circ$  with the axis of the bar, the plane being Perpendicular to one face of the bar.
- 9 At a point of a material the two-dimensional stress system is defined by
$$\sigma_x = 60.0 \text{ MN/m}^2, \text{ tensile}$$
$$\sigma_y = 45.0 \text{ MN/m}^2, \text{ compressive}$$
$$\tau_{xy} = 37.5 \text{ MN/m}^2, \text{ shearing.}$$
10. The state of stress at a point in a loaded structure is given by the strength tensile.

$$T\sigma = \begin{bmatrix} 140 & 60 & 0 \\ 60 & -90 & 0 \\ 0 & 0 & 0 \end{bmatrix}$$

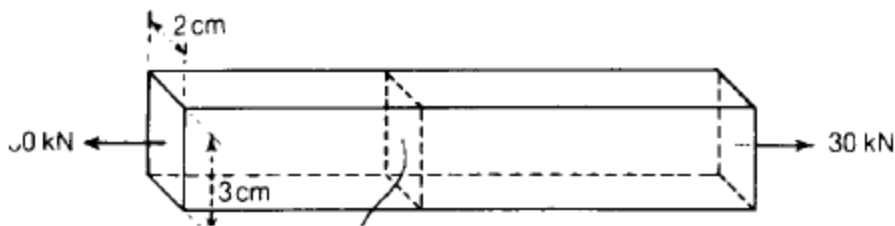
Determine

- The maximum principal stress
- The minimum principal stress
- The maximum and minimum shear strain
- The maximum and minimum shear stress

Taking  $E = 2.1 \times 10^5 \text{ N/mm}^2$      $J = 0.32$

**SOLUTION**

1.



The area of a normal cross-section of the bar is

Tension and compression: direct stresses

$$A = 0.03 \times 0.02 = 0.6 \times 10^{-3} \text{ m}^2$$

The average tensile stress over this cross-section is then

$$B = P/A$$

$$B = 30 \times 10^3 / 0.6 \times 10^{-3} = 50 \text{ MN/m}^2.$$

2. The area of a normal cross-section is

$$A = \pi/4 (0.10)^2 = 7.85 \times 10^{-3} \text{ m}^2$$

The average compressive stress over this cross-section is then

$$B = P/A = 70 \times 10^3 / 7.85 \times 10^{-3} = 8.92 \text{ MN/m}^2$$

The average compressive strain over the length of the cylinder is

$$E = 0.02 \times 10^{-2} / 30 \times 10^{-2} = 0.67 \times 10^{-3}$$

5. The original cross – section of the bar is

$$A_0 = \pi/4 (0.020)^2 = 0.314 \times 10^{-3} \text{ m}^2$$

(i) The average tensile stress at yielding is then

$$B_y = P_y/A_0 = 80 \times 10^3 / 0.314 \times 10^{-3} = 254 \text{ MN/m}^2$$

Where  $P_y$  = load at the yield point.

(ii) The ultimate stress is the normal stress at the maximum load

$$B_{ult} = P_{max}/ A_0 = 150 \times 10^3 / 0.314 \times 10^{-3} = 477 \text{ MN/m}^2$$

Where  $P_{max}$  is maximum load

(iii) The cross-sectional area in the fracture neck is

$$A_F = \pi/4 (0.010)^2 = 0.0785 \times 10^{-3} \text{m}^2$$

The average stress at the breaking point is then

$$\sigma_f = P_f / A_f = 70 \times 10^3 / 0.0785 \times 10^{-3} = 892 \text{MN/m}^2$$

Where  $P_f$  = final breaking load.

8. We have  $\theta = 60^\circ$ ,  $P = 20 \text{KN}$  and  $A = 0.507 \times 10^{-3} \text{m}^2$ . Then

$$\sigma_x = 20 \times 10^3 / 0.507 \times 10^{-3} = 39.4 \text{MN/m}^2$$

The normal stress on the oblique plane is

$$\sigma = \sigma_x \cos^2 60^\circ = (39.4 \times 10^6) / 4 = 9.85 \text{MN/m}^2$$

The shearing stress on the oblique plane is

$$\frac{1}{2} \sigma_x \sin 120^\circ = \frac{1}{2} (39.4 \times 10^6) \frac{\sqrt{3}}{2} = 17.05 \text{MN/m}^2$$

10.

$$\bar{\sigma} = \begin{bmatrix} 140 & 60 & 0 \\ 60 & -90 & 0 \\ 0 & 0 & 0 \end{bmatrix} = \begin{bmatrix} \sigma_x & \tau_{xy} & \tau_{xz} \\ \tau_{yx} & \sigma_y & \tau_{yz} \\ \tau_{zx} & \tau_{zy} & \sigma_z \end{bmatrix}$$

Base on the similarity of matrix

$$\sigma_x = 140 \text{N/mm}^2$$

$$\tau_{xy} = \tau_{yx} = 60 \text{N/mm}^2$$

$$\sigma_y = -90 \text{N/mm}^2$$

$$\text{Maximum Principal Stress } (\sigma_1) = (\sigma_x + \sigma_y)/2 + 1/2 \sqrt{(\sigma_{xx} - \sigma_{yy})^2 + 4\tau_{xy}^2}$$

Substituting into the equation

$$\sigma_1 = (140 - 90)/2 + 1/2 \sqrt{(140 + 90)^2 + 4(60)^2}$$

$$25 + \frac{1}{2} \sqrt{52,900 + 14,400}$$

$$25 + \frac{1}{2} \sqrt{67300}$$

$$25 + \frac{1}{2} \times 259.42$$

$$25 + 129.71$$

$$\sigma_1 = 154.71 \text{N/mm}^2$$

$$\text{Minimum Principal Stress } (\sigma_2) = (\sigma_x + \sigma_y)/2 - 1/2 \sqrt{(\sigma_{xx} - \sigma_{yy})^2 + 4\tau_{xy}^2}$$

Substituting into the equation

$$\sigma_2 = (140 - 90)/2 + 1/2 \sqrt{(140 + 90)^2 + 4(60)^2}$$

$$25 - \frac{1}{2} \sqrt{52,900 + 14,400}$$

$$25 - \frac{1}{2} \sqrt{67300}$$

$$25 - \frac{1}{2} \times 259.42$$

$$25 - 129.71$$

$$\sigma_2 = -104.71 \text{ N/mm}^2$$

#### SHEAR STRESS

$$\tau_{\max} = \frac{1}{2} (\sigma_1 - \sigma_2)$$

Substituting in the formula

$$\tau_{\max} = \frac{1}{2} [(155 - (-105))] = 130 \text{ N/mm}^2$$

$$\tau_{\min} = -\frac{1}{2} [(155 + 105)] = -130 \text{ N/mm}^2$$